

# **Theory of Torsion: A Summary**

Eur Ing **Professor M. Macdonald** BSc MSc PhD CEng FIMechE FIES FHEA **Department Engineering Room M203A** T: 0141 331 3540 E: <u>mmd3@gcu.ac.uk</u>

# **GLASGOW CALEDONIAN UNIVERSITY**

## SCHOOL OF ENGINEERING & BUILT ENVIRONMENT

### ENGINEERING DESIGN & ANALYSIS 2 (M2H721926) – Torsion Theory

#### Torque

Torque is defined as the turning or twisting effect of a force about a point around which it rotates. Referring to the figure, the torque T produced by the tangentially acting force F is the product of the force and the radius r at the point of force application, i.e.



#### **Power and Torque**

If a shaft transmits power at rotational speed of N (rev/min), the torque T (Nm) carried by the shaft is given by:

Power = work done by torque per second

= torque x angular speed

$$\therefore \qquad P = \frac{2\pi NT}{60} \text{ (W)}$$

Alternatively, since  $\omega = \frac{2\pi N}{60}$ , then the transmitted power can be calculated from  $P = T\omega$ 

## **Theory of Torsion**

$$\frac{T}{J} = \frac{G\theta}{L} = \frac{\tau}{r}$$

Other useful arrangements of this formula are as follows:

$$\tau = \frac{T r}{J}, \qquad \theta = \frac{T L}{G J} \qquad \text{and} \qquad \text{stiffness} = \frac{T}{\theta} = \frac{G J_p}{L}$$

Note:  $J = \frac{\pi d^4}{32}$  for a solid shaft,  $J = \frac{\pi (d_2^4 - d_1^4)}{32}$  for a hollow shaft and  $J = 2\pi r^3 t$  for a thin-walled shaft or tube.

Some important points should be noted -

- 1. The angle of twist  $\theta$  varies *directly* with length *l*.
- 2. Since  $\tau = Tr/J$ , for a given torque *T* the shear stress  $\tau$  is proportional to the radius *r*. Thus the maximum shear stress occurs at the outside surface where r = d/2, and the shear stress at the centre of the shaft is zero. The figure shows the variation of  $\tau$  across a diameter.



Variation of au with Radius r.