

#### 4.10. Solved Problems – Belt Drive System Design

##### Problem 1

Calculate the maximum power that may be transmitted by a flat belt driving a pulley 360 mm diameter which rotates at 180 rev/min. The maximum belt tension is 500 N and the angle of lap is 145°. The coefficient of friction between the belt and the pulley is 0.35.

##### Solution

Data:

Pulley dia,  $d = 360 \text{ mm} = 0.36 \text{ m}$

∴  $r = 0.18 \text{ m}$

$N = 180 \text{ rev/min}$

Max. tension (tight side),  $F_1 = 500 \text{ N}$

Angle of lap,  $\theta = 145^\circ$

$\mu = 0.35$

Angle of lap,  $\theta = 145^\circ = (145/360) \times 2\pi = 2.5307 \text{ rads}$

$$\frac{F_1}{F_2} = e^{\mu\theta} \quad \therefore \quad \frac{500}{F_2} = e^{0.35 \times 2.5307} = 2.425 \quad \therefore \quad F_2 = 206.2 \text{ N}$$

$$\text{Power, } P = T\omega = (F_1 - F_2)r\omega = \frac{(F_1 - F_2).r.2\pi N}{60} = \frac{(500 - 206.2) \times 0.18 \times 2\pi \times 180}{60} = \mathbf{997 \text{ W}}$$

##### Problem 2

A multiple vee-belt drive is required to transmit 30 kW from a pulley 160 mm diameter rotating at 450 rev/min. The angle of lap is 170° and the groove angle is 40°. The coefficient of friction between the belt and the pulley is 0.35 and the maximum permissible stress in the belt material is 2.6 MNm<sup>-2</sup>. If the cross-sectional area of each belt is 600 mm<sup>2</sup>, calculate the minimum number of belts required for the drive system.

##### Solution

Data:

Power,  $P = 30 \text{ kW} = 30 \times 10^3 \text{ W}$

Pulley dia,  $d = 160 \text{ mm} = 0.16 \text{ m}$

∴  $r = 0.08 \text{ m}$

$N = 450 \text{ rev/min}$

Angle of groove,  $\alpha = 40^\circ/2 = 20^\circ$

Angle of lap,  $\theta = 170^\circ$

$\mu = 0.35$

Belt material max stress,  $\sigma = 2.6 \text{ MNm}^{-2} = 2.6 \text{ Nmm}^{-2}$

Belt cross-sectional area,  $A_b = 600 \text{ mm}^2$

Angle of lap,  $\theta = 170^\circ = (170/360) \times 2\pi = 2.967 \text{ rads}$

$$\text{Power, } P = T\omega \quad \therefore \quad T = \frac{P}{\omega} = \frac{60P}{2\pi N} = \frac{60 \times (30 \times 10^3)}{2\pi \times 450} = 636.6 \text{ Nm}$$

$$T = (F_1 - F_2)r \quad \therefore F_1 - F_2 = \frac{T}{r} = \frac{636.6}{0.08} = 7958 \text{ N} \quad (1)$$

$$\frac{F_1}{F_2} = e^{\frac{\mu\theta}{\sin\alpha}} \quad \therefore \frac{F_1}{F_2} = e^{\frac{0.35 \times 2.967}{\sin 20^\circ}} = 20.83 \quad \therefore F_2 = \frac{F_1}{20.83} \quad (2)$$

Substitute (2) in (1):  $F_1 - \frac{F_1}{20.83} = 7958 \quad \therefore F_1 - 0.048F_1 = 7958 \quad \therefore 0.952F_1 = 7958$

$$\therefore F_1 = 8359 \text{ N (i.e. total tight side tension)}$$

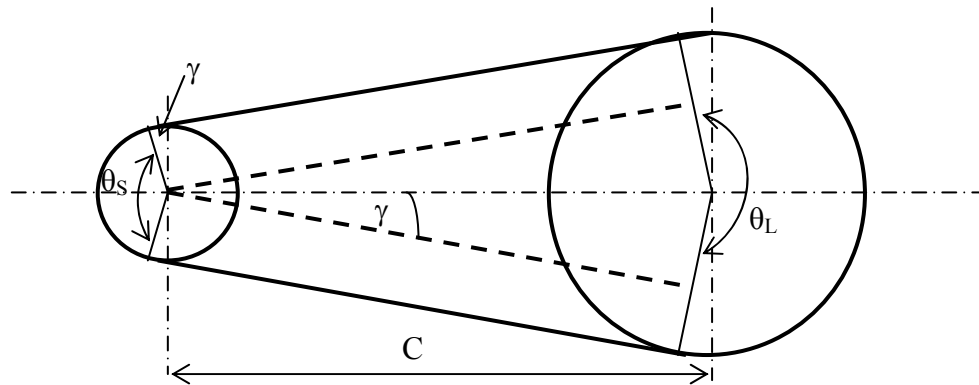
$$\text{Now, } F_1 = \sigma A = 2.6 \times 600 = 1560 \text{ N}$$

$$\text{Minimum number of belts required} = 8359/1560 = 5.36$$

Hence **6 belts** will be required!

### Problem 3

A multiple vee-belt drive is required to transmit 24 kW from a pulley 150 mm in diameter to another pulley 300 mm in diameter rotating at 300 rev/min. The centre distance ( $C$  – see the figure below) is 1000 mm and the groove angle is  $40^\circ$ . The coefficient of friction between the belt and the pulley is 0.31 and the maximum permissible belt tension is 500 N. For this geometry and conditions, calculate the minimum number of belts required for the drive system.



### Solution

Data:

$$\text{Power, } P = 24 \text{ kW} = 24 \times 10^3 \text{ W}$$

$$\text{Small Pulley dia, } d_1 = 150 \text{ mm} = 0.15 \text{ m}$$

$$\therefore r_1 = 0.075 \text{ m}$$

$$\text{Large Pulley dia, } d_2 = 300 \text{ mm} = 0.30 \text{ m}$$

$$\therefore r_2 = 0.15 \text{ m}$$

$$N_2 = 300 \text{ rev/min}$$

$$\text{Angle of groove, } \alpha = 40^\circ/2 = 20^\circ$$

$$\mu = 0.31$$

Belt material max permissible tension,  $F_{\max} = 500 \text{ N}$

Belt angle,  $\gamma$  (from the geometry)

$$\begin{aligned}\sin \gamma &= (r_2 - r_1) / C \\ &= (150 - 75) / 1000 \\ \Rightarrow \gamma &= \sin^{-1}(0.075) \\ &= 4.30^\circ \\ &= 0.075 \text{ rad}\end{aligned}$$

Angle of lap,  $\theta_s = \pi - 2 \cdot \gamma = 2.9915 \text{ rads}$

$$\text{Speed of small pulley, } N_1 = N_2 \times \frac{d_2}{d_1} = 300 \times \frac{0.30}{0.15} = 600 \text{ rpm}$$

$$\text{Power, } P = T\omega \quad \therefore T = \frac{P}{\omega} = \frac{60P}{2\pi N_1} = \frac{60 \times (24 \times 10^3)}{2\pi \times 600} = 381.97 \text{ Nm}$$

$$T = (F_1 - F_2)r \quad \therefore F_1 - F_2 = \frac{T}{r_1} = \frac{381.97}{0.075} = 5092.96 \text{ N} \quad (1)$$

$$\frac{F_1}{F_2} = e^{\frac{\mu\theta}{\sin \alpha}} \quad \therefore \frac{F_1}{F_2} = e^{\frac{0.31 \times 2.9915}{\sin 20^\circ}} = 15.05 \quad \therefore F_2 = \frac{F_1}{15.05} \quad (2)$$

$$\text{Substitute (2) in (1): } F_1 - \frac{F_1}{15.05} = 5092.96 \quad \therefore F_1 - 0.0664F_1 = 5092.96$$

$$\therefore 0.9336F_1 = 5092.96$$

$$\therefore F_1 = 5455.42 \text{ N (i.e. total tight side tension)}$$

$$\text{Minimum number of belts required} = 5455.42 / 500 = 10.91$$

Hence **11 belts** will be required!